

# Deflection and Stress analysis of a Circular plate

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*For questions, please fill out contact form*

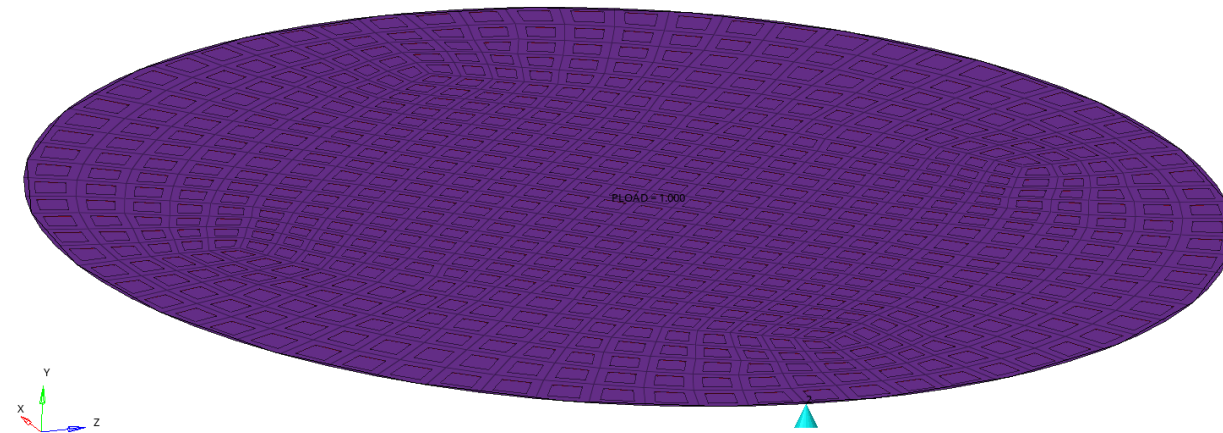


**ALGO**  
**Engineering**  
Simplifying FEA

# Model Description

- Deflection and stress analysis of a circular plate based on example question 1 documented in page 429 of the following book :
  - J. Souza, *Roark's Formulas For Stress And Strain-.pdf*. Accessed: Nov. 06, 2022. [Online]. Available: [https://www.academia.edu/37205286/Roarks\\_Formulas\\_For\\_Stress\\_And\\_Strain\\_pdf](https://www.academia.edu/37205286/Roarks_Formulas_For_Stress_And_Strain_pdf)
- As mentioned in the book, plate is steel

Model Info: C:/ALGO\_FEA\_MODELS/0000\_FREE\_MODELS/Circular plate/Run 3/CP\_model.hm\*



# Model Description

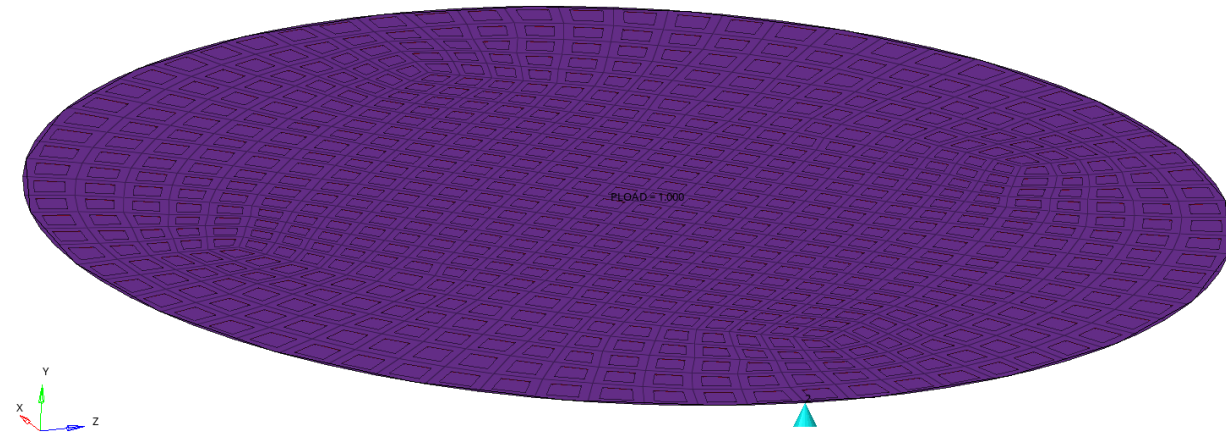
Diameter – 20 in (508 mm)

Thickness – 0.2 in (5.08 mm)

Young's modulus – 30000000 lb/in<sup>2</sup>  
(206.8427184 GPa)

Poisson's ratio – 0.285

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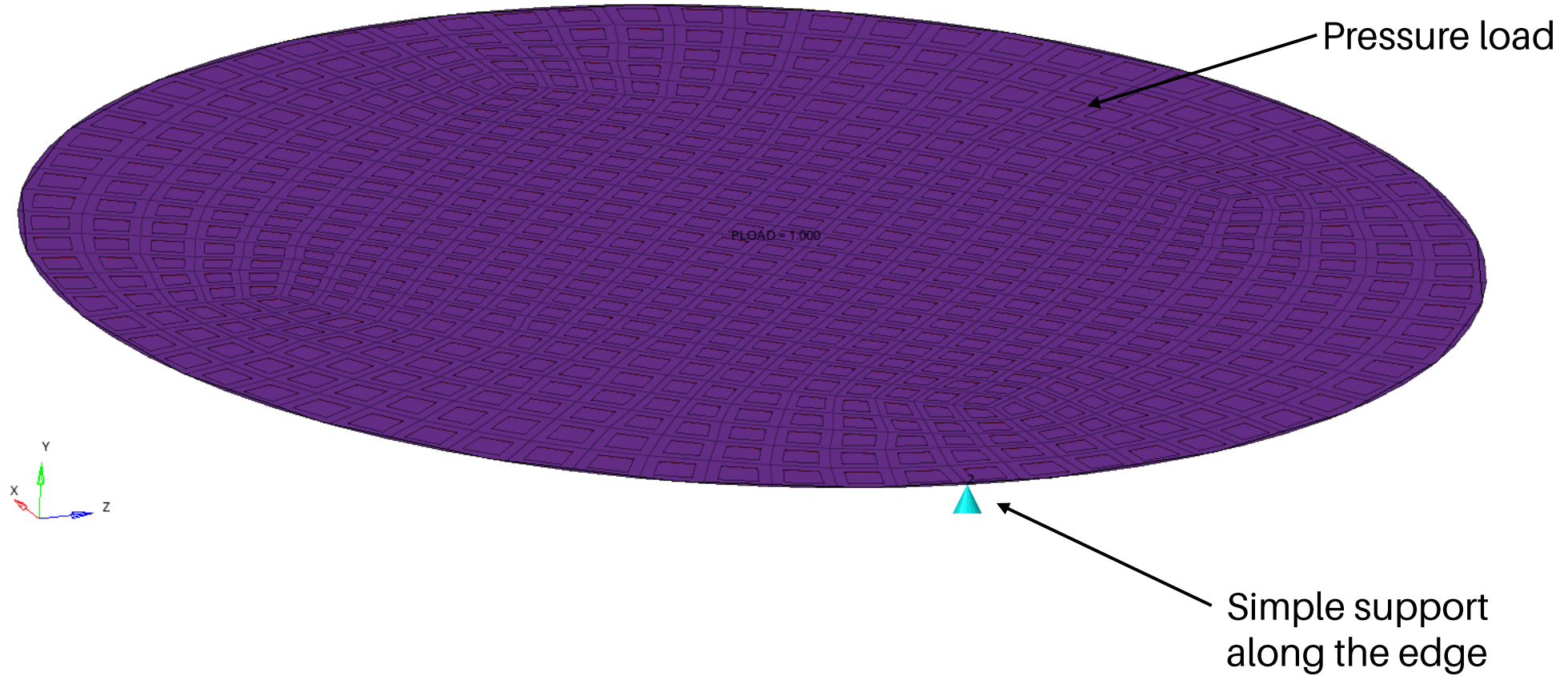
# Model Parameters

Entity	Type
Solver	Altair Radioss
Version	2021.2.1
Processors	2
Threads	2
CPU	Intel(R) Core(TM) i7-9750H CPU @ 2.60GHz
Total run time	1 sec

FEA Entities	Type
Analysis Type	Dynamic Explicit
Unit System	Kg, mm, ms
Element Type	SHELL4N
Material Type	M1_ELAST
Property Type	P1_SHELL

# Analysis Setup

Model Info: C:/ALGO\_FEA\_MODELS/0000\_FREE\_MODELS/Circular plate/Run 3/CP\_model.hm\*



# Analysis Assumptions and Limitations

- Standard density value of steel is assumed.
- Distributed load is assumed as a pressure load.

# Hand Calculations

$D$  – Plate constant     $E$  – Young's modulus     $\nu$  – Poisson's ratio     $t$  – thickness  
 $q$  – load per unit area     $r_o$  – start of a distributed load     $a$  – outer radius

$$D = \frac{Et^3}{12(1-\nu^2)} = 218000 \quad \text{then, } q = 3 \quad r_o = 10$$

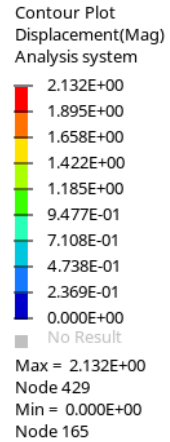
$$\text{Center deflection} = \frac{qa^4(5+\nu)}{64D(1+\nu)} = 0.0833 \text{ in}$$

$$\text{Maximum bending moment}(M) = \frac{qa^3}{16}(3+\nu) = 61.5 \text{ lb-in/in}$$

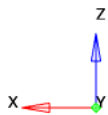
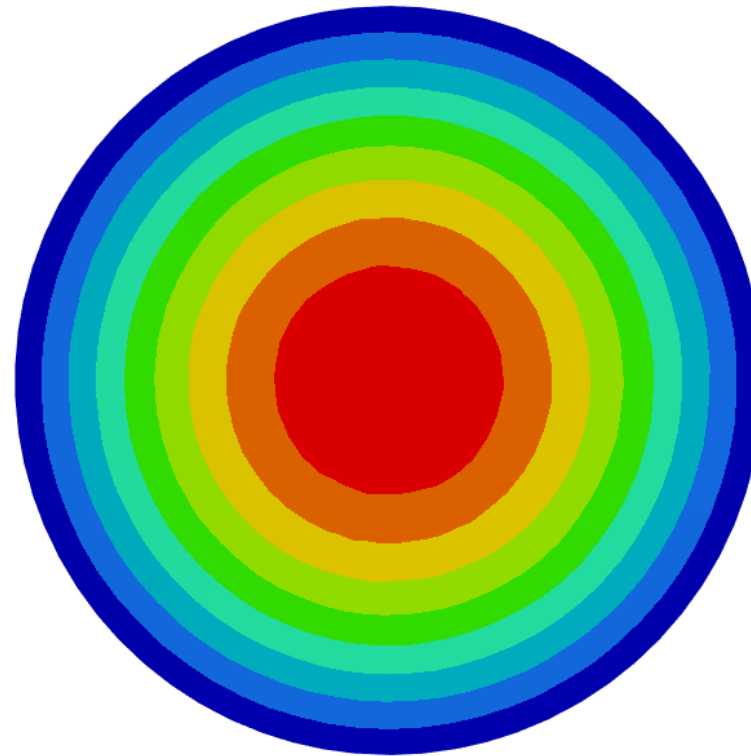
$$\text{Maximum stress} = \frac{6M}{t^2} = 9240 \text{ lb/in}^2$$

# Analysis Results – Center Deflection

Units - mm



1: CP\_model  
Loadcase 1 : Time = 1.0000e+03 : Frame 101





# Analysis Results – Maximum Stress

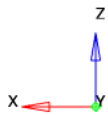
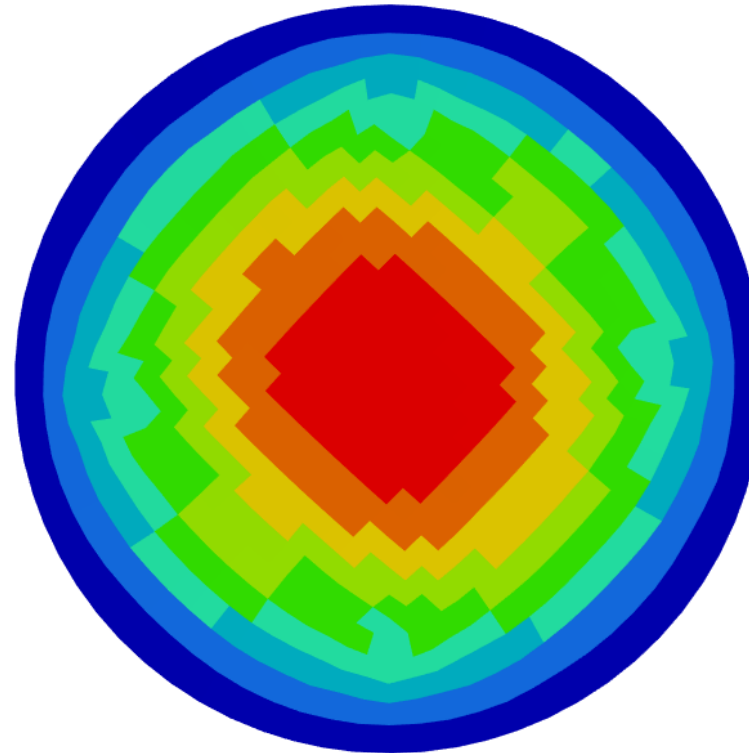
Units - GPa

Contour Plot  
Stress(YY, Max)  
Analysis system

6.385E-02
5.926E-02
5.467E-02
5.008E-02
4.549E-02
4.091E-02
3.632E-02
3.173E-02
2.714E-02
2.255E-02
No Result

Max = 6.385E-02  
SHELL 643  
Min = 2.255E-02  
SHELL 4

1: CP\_model  
Loadcase 1 : Time = 1.0000e+03 : Frame 101



# Analysis Results

- Center Deflection,  
Based on hand calculations – 0.0833 in (2.1158 mm)  
From the simulation - 0.0839 in (2.132 mm)
- Maximum Stress,  
Based on hand calculations – 9240 lb/in<sup>2</sup> (63.708 MPa)  
From the simulation - 9260.66 lb/in<sup>2</sup> (63.85 MPa)

# Conclusions

- Deflection analysis of a circular plate conducted using Altair Radioss based on the book listed in slide 2.
- Results of the simulation correlate well to the expected hand calculation value.